# Experimental Analysis of Buried High Density Polyethylene Pipes 

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Received 05.09.2000


#### Abstract

The primary goal of this study was to evaluate the deflections and bending moments of buried High density polyethylene (HDPE) pipes. The performance of buried HDPE pipes subjected to live loading was studied in soil chambers for three levels of service loading. Strains and diametral changes were measured for 10,000 hours. The discussion of the experimental findings is focused on certain recent concerns, associated with the HDPE piping related to deflection, longitudinal, and transverse stresses and bending moments. A $7.5 \%$ vertical change of diameter, which is the failure criterion, was observed at approximately 3,200 hours for the specimens heated at $50^{\circ} \mathrm{C}$, and subjected to maximum service loading.


Key words: Infrastructure, Pipeline, Pipe-soil interaction, Viscoelasticity, HDPE

## Introduction

The high-density polyethylene pipe has good potential for economic use for marine oil and gas pipelines, underdrains, storm sewers, culverts, and other subsurface drainage structures. In view of its inherent chemical and corrosion resistance, light weight, toughness, flexibility, easy splicing, and consequent easy handling and installation, HDPE piping is used extensively for gas pipelines. In the transportation industry, over forty states in the US use HDPE pipes as part of a $40 \%$ annual growth in the use of thermoplastic, HDPE and polyvinyl chloride (PVC) pipes in transportation construction projects (Goddard, 1995).

## Experiments

Two types of corrugated HDPE pipe specimens of nominal inside diameters 305 mm were considered. Although there were small geometrical property differences between the two pipes, both have the same
cell classification. Specifications of the materials are as follows:

Melt index: 0.4-0.15
Section area: $4.78-\mathrm{mm}^{2} / \mathrm{mm}$
Cell classification: 335420C
Inside diameter: 305 mm (12 in.)
Outside diameter: 360.68 mm (14.2 in.)
Modulus of the pipe: 758 MPa ( 110 ksi )
Thickness (single wall): 3.2 mm (0.129 in.)
Weight $[\mathrm{kg} / 6 \mathrm{~m}(\mathrm{lb} / 20 \mathrm{ft})]: 28.96 \mathrm{~kg}(63.8 \mathrm{lb})$
Type of soil: ASTM D2321 Class II, SW/SP
Color and UV stabilizer: Black, $2 \%$ min. carbon
Density: $0.941,0.955 \mathrm{~g} / \mathrm{cm}^{3}(0.03397,0.03448$ $\mathrm{lb} / \mathrm{in}^{3}$ )

Length of the pipe: $1981.2 \mathrm{~mm}\left(6^{\prime} 6^{\prime \prime}\right)$ corrugated (annular)

Tensile strength: $552-758 \mathrm{MPa}$ (80000110000 psi ) at yield

Moment of inertia: $0.574-0.522 \mathrm{~cm}^{4} / \mathrm{cm}(0.035-$ 0.0319 in. ${ }^{4} / \mathrm{in}$.) Type I-II

Load levels: M (maximum service load) $=5600 \mathrm{lb}$, $3700 \mathrm{lb}(2 / 3 \mathrm{M})$ and $1900 \mathrm{lb}(1 / 3 \mathrm{M})$

The experimental investigation addressed the changes in diameter and the strains of Type I and II buried pipes subjected to AASHTO loading. The long-term behavior was accelerated with superambient temperatures of 40 and $50^{\circ} \mathrm{C}$ to provide the data for life prediction at the ambient temperature of $20^{\circ} \mathrm{C}$ based on $7.5 \%$ vertical deflection as the failure criterion. Hence, readings were taken up to failure or 10,000 hours. The supports at the end of the pipe were located so that the bending effects could be eliminated.

## Soil Chambers and Installation of the Measuring Devices

Each of the seven soil chambers was of 914.4 mm depth, 1980 mm length, and 3660 mm width. Every chamber was divided into four parts to accommodate four pipe specimens as shown in Figures 1 and 2. Pairs of specimens, notched (N) [Notch depth and length: 0.635 mm ( 0.025 in .) and 7.94 mm ( $5 / 16 \mathrm{in}$.), ASTM 1474] and un-notched (U), were loaded simultaneously by using 2 channels $610 \mathrm{~mm} \times 250 \mathrm{~mm} \times 50$ mm , to simulate a typical tire footprint, and a steel plate to distribute the load evenly, as shown in Figure 3. Four dial gages were mounted on the guide tubes of all specimens to measure the vertical and horizontal changes in the inside diameter at the mid-section, as shown in Figure 4. Only a few specimens were mounted with two strain gages, as shown in Figure 5 , at the shoulders, located at $45^{\circ}$ and $135^{\circ}$ circumferentially and longitudinally, respectively. These locations correspond to the maximum stress locations (Ataoglu and Reddy, 2001, a-b). A third gage to measure longitudinal strain was located at the bottom.

## Results of the Experimental Investigation

The experimental deflection data are presented for the pipes under different loadings and temperatures in Figures 6 to 19.

The experimental strain data and stresses are presented for Type I/II un-notched pipes under different loadings and temperatures in Reddy et al. (2001). Bending moments ( $\mathrm{M}_{0}, \mathrm{lb}$ in.) values are presented in Tables 1 to 7.


Figure 1. End view of the test set-up


Figure 2. Soil chambers (half full) after compaction


Figure 3. Set-up of structural sections for footprint loading


Figure 4. Set-up of dial gages


Figure 5. Reading of strain gages


Figure 6. Change in diameter of Pipe 1


Figure 7. Change in diameter of Pipe 2


Figure 8. Change in diameter of Pipes 3 and 4


Figure 9. Change in diameter of Pipes 5 and 6

Table 1. Moments for Pipes 23, Load Applied (M), $40^{\circ} \mathrm{C}$, I, U

| Days | $\mathrm{M}_{0}$ <br> $\left(45^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(135^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(270^{\circ}\right)$ |
| :---: | :---: | :---: | :---: |
| 1 | 0 | 0 | 0 |
| 6 | 0.292 | 228.44 | 131.7 |
| 9 | 1.843 | 321.58 | 138 |
| 16 | 3.011 | 419.15 | 141.1 |
| 19 | 4.48 | 530.61 | 151.3 |
| 24 | 5.71 | 654.59 | 159.7 |
| 27 | 6.151 | 713.21 | 193.4 |
| 39 | 7.14 | 906.01 | 234 |
| 41 | 7.758 | 1028.8 | 273.8 |
| 48 | 8.499 | 1080.6 | 324.5 |
| 56 | 9.207 | 1225 | 357.7 |
| 63 | 10.06 | 1279.5 | 436.2 |
| 72 | 10.91 | 1292.6 | 511.3 |
| 80 | 16.19 | 1335.2 | 556.8 |
| 88 | 18.84 | 1357.2 | 592.8 |
| 97 | 22.71 | 1473.7 | 623.8 |
| 103 | 24.68 | 1512.7 | 664.7 |
| 119 | 26.58 | 1565 | 735.3 |
| 126 | 28.74 | 1581.3 | 778.2 |
| 134 | 30.79 | 1655.1 | 833.4 |
| 149 | 31.72 | 1742.5 | 896.7 |
| 159 | 33.69 | 1828.5 | 946.8 |
| 185 | 35.18 | 1879.4 | 973.5 |
| 214 | 35.23 | 1818.3 | 1048 |
| 225 | 35.37 | 1906.2 | 1099 |
| 248 | 34.64 | 1876.4 | 1146 |
| 260 | 34.46 | 1924.7 | 1185 |
| 270 | 34.9 | 1956.9 | 1276 |
| 317 | 32.93 | 1864.9 | 1224 |
| 368 | 30.91 | 1756.8 | 1012 |
|  |  |  |  |
| 1 |  |  |  |



Figure 10. Change in diameter of Pipes 7 and 8

Table 2. Moments for Pipes 24, Load Applied (M), $40^{\circ} \mathrm{C}$, I,U

| Days | $\mathrm{M}_{0}$ <br> $\left(45^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(135^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(270^{\circ}\right)$ |
| :---: | :---: | :---: | :---: |
| 1 | 0 | 0 | 0 |
| 6 | 7.03 | 196.24 | 122.3 |
| 9 | 8.561 | 289.39 | 226.5 |
| 16 | 9.683 | 387.2 | 177 |
| 19 | 11.13 | 498.78 | 208.3 |
| 24 | 12.33 | 623.01 | 261.4 |
| 27 | 12.75 | 681.63 | 294.8 |
| 39 | 13.66 | 874.92 | 334.1 |
| 41 | 14.26 | 997.56 | 373.6 |
| 48 | 14.96 | 1049.8 | 423.7 |
| 56 | 15.61 | 1194.4 | 456 |
| 63 | 16.41 | 1249.1 | 533.9 |
| 72 | 17.21 | 1262.5 | 608 |
| 80 | 22.43 | 1305.4 | 652.6 |
| 88 | 25.04 | 1327.5 | 687.9 |
| 97 | 28.84 | 1444.4 | 717.9 |
| 103 | 30.77 | 1483.6 | 758.2 |
| 119 | 32.57 | 1536.4 | 827.2 |
| 126 | 34.68 | 1552.9 | 869.4 |
| 134 | 36.67 | 1627 | 923.8 |
| 149 | 37.51 | 1714.8 | 985.5 |
| 159 | 39.41 | 1801.2 | 1035 |
| 185 | 40.72 | 1852.8 | 1059 |
| 214 | 40.59 | 1792.6 | 1131 |
| 225 | 40.65 | 1880.8 | 1181 |
| 248 | 39.76 | 1851.8 | 1225 |
| 260 | 39.51 | 1900.5 | 1262 |
| 270 | 39.88 | 1933.1 | 1352 |
| 317 | 37.6 | 1842.5 | 1295 |
| 368 | 35.24 | 1736.1 | 1078 |
|  |  |  |  |



Figure 11. Change in diameter of Pipes 9 and 10


Figure 12. Change in diameter of Pipes 11 and 12

Table 3. Moments for Pipe 8, Load Applied (M), $50^{\circ} \mathrm{C}$, I, U

| Days | $\mathrm{M}_{0}$ <br> $\left(45^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(135^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(270^{\circ}\right)$ |
| :---: | :---: | :---: | :---: |
| 1 | 0.00 | 0 | 0 |
| 3 | 0.56 | 242.59 | 56.604 |
| 5 | 1.41 | 457.32 | 107.61 |
| 13 | 1.79 | 629.86 | 186.82 |
| 28 | 6.75 | 880.74 | 315.49 |
| 38 | 9.74 | 1014.9 | 442.43 |
| 45 | 11.53 | 1188.6 | 620.15 |
| 56 | 18.68 | 1354.0 | 651.47 |
| 79 | 26.59 | 1683.2 | 649.62 |
| 87 | 33.58 | 1922.3 | 729.39 |
| 91 | 38.03 | 2197.2 | 824.73 |
| 101 | 41.13 | 2246.5 | 840.19 |
| 124 | 42.65 | 2343.3 | 874.76 |
| 148 | 43.95 | 2368.3 | 1016.7 |
| 188 | 46.73 | 2347.7 | 1058.1 |



Figure 13. Change in diameter of Pipes 13 and 14

Table 4. Moments for Pipe 17, Load Applied (M), $20^{\circ} \mathrm{C}$, II, U

| Days | $\mathrm{M}_{0}$ <br> $\left(45^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(135^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(270^{\circ}\right)$ |
| :---: | :---: | :---: | :---: |
| 1 | 0.00 | 0.00 | 0.00 |
| 6 | 1.26 | 226.51 | 54.68 |
| 9 | 2.75 | 334.87 | 176.00 |
| 16 | 3.87 | 448.56 | 119.10 |
| 19 | 5.28 | 578.29 | 155.75 |
| 24 | 6.45 | 722.70 | 217.88 |
| 27 | 6.88 | 790.89 | 257.00 |
| 39 | 7.82 | 1015.56 | 303.76 |
| 41 | 8.41 | 1158.28 | 349.90 |
| 48 | 9.11 | 1218.86 | 408.78 |
| 56 | 9.79 | 1387.02 | 446.99 |
| 63 | 10.60 | 1450.64 | 538.09 |
| 72 | 11.41 | 1466.18 | 625.03 |
| 80 | 16.48 | 1516.06 | 677.66 |
| 88 | 19.03 | 1541.90 | 719.26 |
| 97 | 22.73 | 1677.69 | 754.96 |
| 103 | 24.62 | 1723.27 | 802.31 |
| 119 | 26.44 | 1784.75 | 884.05 |
| 126 | 28.50 | 1803.91 | 933.63 |
| 134 | 30.47 | 1890.09 | 997.59 |
| 149 | 31.36 | 1992.19 | 1070.55 |
| 159 | 33.24 | 2092.61 | 1128.52 |
| 185 | 34.64 | 2152.76 | 1158.69 |
| 214 | 34.67 | 2082.83 | 1244.98 |
| 225 | 34.79 | 2185.35 | 1303.86 |
| 248 | 34.06 | 2151.73 | 1357.61 |
| 260 | 33.88 | 2208.33 | 1402.00 |
| 270 | 34.29 | 2246.20 | 1507.09 |
| 317 | 32.35 | 2140.87 | 1446.01 |
| 368 | 30.36 | 2017.35 | 1197.36 |
|  |  |  |  |
| 20 |  |  |  |



Figure 14. Change in diameter of Pipes 15 and 16

Table 5. Moments for Pipe 18, Load Applied (M), $20^{\circ} \mathrm{C}$, II, U

| Days | $\mathrm{M}_{0}$ <br> $\left(45^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(135^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(270^{\circ}\right)$ |
| :---: | :---: | :---: | :---: |
| 1 | 0 | 0 | 0 |
| 6 | 0.99 | 151.53 | 15.62 |
| 9 | 2.479 | 260.11 | 23.36 |
| 16 | 3.598 | 374.31 | 27.84 |
| 19 | 5.009 | 504.27 | 40.09 |
| 24 | 6.189 | 649.05 | 50.64 |
| 27 | 6.611 | 717.46 | 90.26 |
| 39 | 7.554 | 943.02 | 139 |
| 41 | 8.147 | 1085.9 | 185.5 |
| 48 | 8.854 | 1147 | 245.6 |
| 56 | 9.53 | 1315.7 | 285.1 |
| 63 | 10.34 | 1379.9 | 377.4 |
| 72 | 11.16 | 1396.1 | 465.8 |
| 80 | 16.23 | 1446.6 | 519.8 |
| 88 | 18.78 | 1473 | 562.8 |
| 97 | 22.49 | 1609.4 | 600 |
| 103 | 24.38 | 1655.5 | 648.3 |
| 119 | 26.2 | 1718.1 | 732.8 |
| 126 | 28.27 | 1737.8 | 783.5 |
| 134 | 30.23 | 1824.6 | 848.8 |
| 149 | 31.12 | 1927.8 | 924.3 |
| 159 | 33.01 | 2029 | 984 |
| 185 | 34.42 | 2091 | 1019 |
| 214 | 34.46 | 2023.2 | 1110 |
| 225 | 34.58 | 2126.6 | 1170 |
| 248 | 33.86 | 2094.7 | 1228 |
| 260 | 33.68 | 2152.2 | 1274 |
| 270 | 34.1 | 2190.8 | 1381 |
| 317 | 32.94 | 2088.9 | 1328 |
| 368 | 32.38 | 1969.2 | 1088 |
|  |  |  |  |

Figure 16. Change in diameter of Pipes 19 and 20


Figure 17. Change in diameter of Pipes 21 and 22


Figure 18. Change in diameter of Pipes 23 and 24


Figure 19. Change in diameter of Pipes 25 and 26

Table 6. Moments for Pipe 1, Load Applied (M/3), $50^{\circ} \mathrm{C}, \mathrm{II}, \mathrm{U}$

| Days | $\mathrm{M}_{0}$ <br> $\left(45^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(135^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(270^{\circ}\right)$ |
| :---: | :---: | :---: | :---: |
| 1 | 0 | 0 | 0 |
| 3 | 0.15 | 21.93 | 114.37 |
| 5 | 0.48 | 131.35 | 187.64 |
| 13 | 3.08 | 394.04 | 201.67 |
| 28 | 4.21 | 449.29 | 209.36 |
| 38 | 5.63 | 780.60 | 211.79 |
| 45 | 6.49 | 590.28 | 310.91 |
| 56 | 8.02 | 613.31 | 326.70 |
| 79 | 8.52 | 660.99 | 358.03 |
| 87 | 9.52 | 685.54 | 416.78 |
| 91 | 10.23 | 734.11 | 453.63 |
| 101 | 11.62 | 772.91 | 509.61 |
| 148 | 12.12 | 813.90 | 573.49 |
| 162 | 12.62 | 876.14 | 594.66 |
| 188 | 12.36 | 922.58 | 586.86 |

The maximum effective stress was 2.614 MPa (379.17 psi), (i.e. $7.5 \%$ deflection of the diameter), which is much less than 20.68 MPa ( 3000 psi ), which is the Corrugated Polyethylene Pipe Association (CPPA) yield stress (Reddy et al., (2001). The change in diameter is the governing factor and the CPPA limit is not reasonable for the general failure criterion of the buried HDPE pipe subjected to live loading.

## Conclusions

The discussion of the experimental findings is focused on the HDPE piping related to deflections, stresses, and bending moments. H-20 truck loading was used to determine the maximum allowable loading of the specimens. The actual loading of most of the vehicles is less severe than that for the $\mathrm{H}-20$ truck specification. In view of the strong time and temperature dependence of polyethylene, exposure to super-ambient temperatures was used to accelerate the failure mechanisms for service life prediction of the viscoelastic HDPE pipe. A $7.5 \%$ vertical change in diameter (the failure criterion) and bending were observed. A $7.5 \%$ vertical change in diameter was observed for the specimens heated to $50^{\circ} \mathrm{C}$, with the maximum loading. Life prediction was determined from the Arrhenius equation and the $\mathrm{Bi}-$ directional Shifting Function method (BSM). Both methods gave similar life predictions, as shown in Table 8, but the BSM was more conservative (Reddy et al., 2001). A $7.5 \%$ vertical change in diameter, or more, was observed at approximately 3,200 hours for the specimens heated at $50^{\circ} \mathrm{C}$, and subjected to maximum loading. A 6 to $7 \%$ vertical change of diameter was observed at 10,000 hours for the specimens heated at $40^{\circ} \mathrm{C}$ and subjected to maximum loading. Therefore, extrapolation had to be performed for this temperature environment to determine the corresponding time of failure ( $7.5 \%$ vertical change in diameter). From these values, life prediction at ambient temperatures $\left(20^{\circ} \mathrm{C}\right)$, based on vertical changes of diameter, was performed. Notches accelerated the vertical changes in diameter, but no creep-rupture was observed within the time frame of 10,000 hours. The maximum service lives for specimens at ambient temperature and subjected to maximum loading were approximately 30 and 80 years for notched/unnotched specimens, respectively, assuming proper installation and $90 \%$ compaction.

Table 7. Moments for Pipe 7, Load Applied (M), $50^{\circ} \mathrm{C}$, II,U

| Days | $\mathrm{M}_{0}$ <br> $\left(45^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(135^{\circ}\right)$ | $\mathrm{M}_{0}$ <br> $\left(270^{\circ}\right)$ |
| :---: | :---: | :---: | :---: |
| 1 | 0.00 | 0.00 | 0.00 |
| 3 | 0.21 | 81.47 | 50.14 |
| 5 | 1.02 | 193.90 | 131.35 |
| 13 | 1.40 | 260.63 | 245.12 |
| 28 | 6.18 | 327.04 | 409.57 |
| 38 | 9.05 | 521.92 | 521.92 |
| 45 | 10.77 | 632.34 | 735.98 |
| 56 | 17.66 | 871.71 | 744.00 |
| 79 | 25.28 | 1098.75 | 755.21 |
| 87 | 32.00 | 1329.41 | 840.76 |
| 91 | 36.28 | 1438.18 | 944.48 |
| 101 | 39.26 | 1769.50 | 1111.25 |
| 148 | 40.74 | 1948.81 | 1193.99 |
| 162 | 41.99 | 2175.80 | 1316.22 |
| 188 | 44.68 | 2152.70 | 1307.00 |

For HDPE piping, the yield stress should not exceed $20.68 \mathrm{MPa}(3000 \mathrm{psi})$. Test results indicated that the maximum stress is at the shoulder, and is much less than the CPPA limit (Reddy et al., 2001).

The CPPA limit ( 3000 psi ), which is based on yielding due to the bending, is not reasonable for
the general failure criterion of the buried HDPE pipe subjected to live loading. The deflection threshold seems to be the governing failure criterion.

Table 8. Comparison of the Arrhenius and the Bidirectional Methods

| Type | Arrhenius <br> (years) | Bi-directional <br> (years) |
| :---: | :---: | :---: |
| I-NM | 32.6 | 31.9 |
| I-UM | 81.5 | 91.3 |
| I-U1/3 | 1,141 | 1,712 |
| II-NM | 25.4 | 28.5 |
| II-UM | 76.1 | 78.7 |
| II-U1/3 | 1,130 | $1,027.4$ |

## Acknowledgements

The authors would like to thank the Florida Department of Transportation (Contract Monitor: Mr. R. G. Powers, Material Division) for financial support. Thanks are also due to Dr. S.E. Dunn, Professor and former Chairman, Department of Ocean Engineering, and Dr. J.S. Jurewicz, Dean of Engineering, Florida Atlantic University, for their support and encouragement.

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